

## Designing Two-Phase Devices for Performance

What Physics Rewards. What It Punishes.

Performance is not optimized — it is allocated.  
 Every constraint spends margin somewhere.  
 If you don't know where, physics will decide for you.

### Geometry & Sizing

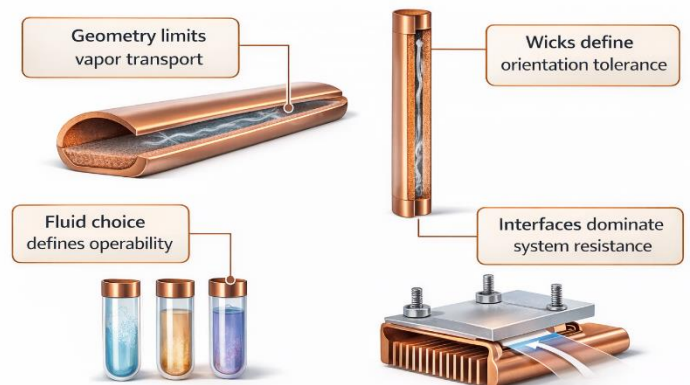
Performance starts with cross-section.

- Cross-section is the highest-value asset.
- Thickness beats width. Every time.
- Short paths outperform clever spreading.

### Punishments

- Heat spreading does not replace vapor area.
- Minimum envelopes do not produce high performance.
- Late packaging changes always cost margin.

### Common Two-Phase Design Mistakes



### Reality check

A 20–30% hit in thickness commonly changes the governing limit.

### Flattening & Forming

Flattening is a trade, not a win.

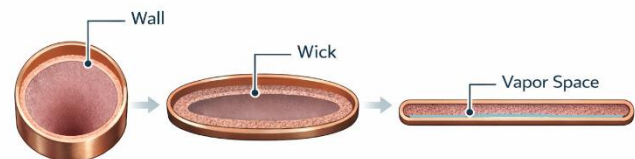
- Contact improves.
- Vapor and liquid-return margin shrink.
- Wick damage is real.

### Punishments

- Flattening does not increase capacity.
- Round-tube assumptions lie once you flatten.

### Rule

Flattened height  $\leq 50\text{--}60\%$  of OD is where vapor transport and wick effects take over.



Flattening increases contact area but reduces vapor volume and wick permeability.

Transport Capacity ↓ → Packaging Efficiency ↑

Figure 2-X. Geometry-driven tradeoffs in flattened heat pipes.

Progressive flattening improves packaging and interface contact but reduces vapor volume and effective wick permeability, limiting maximum heat transport.

### Wick & Liquid Return



LinkedIn



YouTube

Capillary limit kills more designs than anything else.

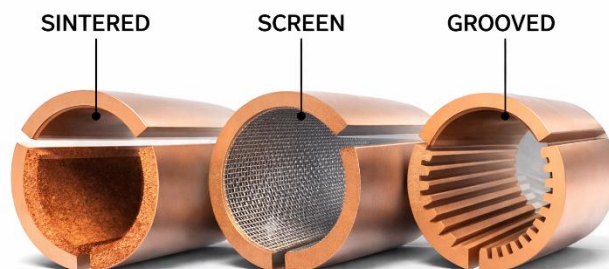
- Orientation matters — even when you wish it didn't.
- Capillary pressure without flow margin still fails.

**Punishments**

- Ideal wick properties do not survive reality.
- Manufacturing, bending, and forming reduce permeability.

**Truth**

In real hardware, the capillary limit is often the governing limit.



**Vapor Space & Transport**

Vapor space is not optional.

- Vapor cross-section sets capacity.
- $\Delta P$  rises fast with flattening and bending.

**Punishments**

- Vapor space treated as leftover volume disappears first.
- Transport limits dominate long, thin, flat geometries.

**Truth**

Long + thin + flat becomes vapor-limited. Always.

**Bending & Routing**

Every bend spends margin.

- Bends ovalize tubes.
- Wicks distort.
- Vapor area shrinks.

**Punishments**

- Straight-tube performance does not survive routing.
- Stacked bends behave like restrictions.

**Rules**

- $R \geq 5 \times OD \rightarrow$  minimal penalty
- $R \geq 3 \times OD \rightarrow$  measurable penalty
- $R < 3 \times OD \rightarrow$  expect degradation



- Bends within ~5xOD compound losses
- Bends near the evaporator are high risk

## Length & Aspect Ratio

Length eats capillary pressure for lunch.

- Transport penalties scale faster than length alone.
- Capillary head is finite — length consumes it directly.

## Punishments

- Short-device data does not extrapolate.
- Condenser area does not save long evaporators.

## Truth

If L/OD looks uncomfortable on the drawing, it already is.

## Heat Input & Evaporator

The hottest square centimeter sets the ceiling.

- Heat input is non-uniform unless proven otherwise.
- Total watts is a distraction.

## Punishments

- Sizing by average power fails.
- Trading transport margin for evaporator resistance backfires.

## System Reality

Most systems are not heat-pipe-limited.

- Airflow, fins, and interfaces often dominate  $\Delta T$ .
- Component optimization  $\neq$  system performance.

## Punishment

- Chasing small two-phase gains while ignoring airflow is theater.

## Final Rule

Vapor cross-section sets capacity.

Liquid return sets robustness.

Everything else is tuning.

Break these rules and the device may still work —  
but it will not be a high-performance solution.

